

# Exam 2 Summary

## Notes

The exam will cover material from Section 3.1 to 3.8. Recall that, for Variation of Parameters, the system of equations will be provided to you:

$$\begin{aligned}u_1' y_1 + u_2' y_2 &= 0 \\u_1' y_1' + u_2' y_2' &= g(t)\end{aligned}$$

So you'll need to remember what all the notation meant (in terms of the original problem).

## Structure and Theory (Mostly 3.2)

The goal of the theory was to establish the structure of solutions to the second order IVP:

$$y'' + p(t)y' + q(t)y = g(t), \quad y(t_0) = y_0$$

We saw that two functions form a fundamental set of solutions to the homogeneous DE if the Wronskian is not zero at  $t_0$ .

1. Vocabulary: Linear operator, general solution, fundamental set of solutions, linear combination.

2. Theorems:

- Abel's Theorem.

If  $y_1, y_2$  are solutions to  $y'' + p(t)y' + q(t)y = 0$ , then the Wronskian,  $W(y_1, y_2)$ , is either always zero or never zero on the interval for which the solutions are valid.

That is because the Wronskian may be computed as:

$$W(y_1, y_2)(t) = Ce^{-\int p(t) dt}$$

- The Structure of Solutions to  $y'' + p(t)y' + q(t)y = g(t)$ ,  $y(t_0) = y_0, y'(t_0) = v_0$   
Given a fundamental set of solutions to the homogeneous equation,  $y_1, y_2$ , then there is a solution to the initial value problem, written as:

$$y(t) = C_1 y_1(t) + C_2 y_2(t) + y_p(t)$$

where  $y_p(t)$  solves the non-homogeneous equation.

In fact, if we have:  $y'' + p(t)y' + q(t)y = g_1(t) + g_2(t) + \dots + g_n(t)$ , we can solve by splitting the problem up into smaller problems:

- $y_1, y_2$  form a fundamental set of solutions to the homogeneous equation.
- $y_{p_1}$  solves  $y'' + p(t)y' + q(t)y = g_1(t)$
- $y_{p_2}$  solves  $y'' + p(t)y' + q(t)y = g_2(t)$   
and so on..
- $y_{p_n}$  solves  $y'' + p(t)y' + q(t)y = g_n(t)$

and the full solution is:  $y(t) = C_1 y_1 + C_2 y_2 + y_{p_1} + y_{p_2} + \dots + y_{p_n}$ .

# Finding the Homogeneous Solution

We had two distinct equations to solve-

$$ay'' + by' + cy = 0 \quad \text{or} \quad y'' + p(t)y' + q(t)y = 0$$

First we look at the case with constant coefficients, then we look at the more general case.

## Constant Coefficients

To solve

$$ay'' + by' + cy = 0$$

we use the **ansatz**  $y = e^{rt}$ . Then we form the associated **characteristic equation**:

$$ar^2 + br + c = 0 \quad \Rightarrow \quad r = \frac{-b \pm \sqrt{b^2 - 4ac}}{2a}$$

so that the solutions depend on the discriminant,  $b^2 - 4ac$  in the following way:

- $b^2 - 4ac > 0 \Rightarrow$  two distinct real roots  $r_1, r_2$ . The general solution is:

$$y_h(t) = c_1 e^{r_1 t} + c_2 e^{r_2 t}$$

If  $a, b, c > 0$  (as in the Spring-Mass model) we can further say that  $r_1, r_2$  are negative. We would say that this system is OVERDAMPED.

- $b^2 - 4ac = 0 \Rightarrow$  one real root  $r = -b/2a$ . Then the general solution is:

$$y_h(t) = e^{-(b/2a)t} (C_1 + C_2 t)$$

If  $a, b, c > 0$  (as in the Spring-Mass model), the exponential term has a negative exponent. In this case (one real root), the system is CRITICALLY DAMPED.

- $b^2 - 4ac < 0 \Rightarrow$  two complex conjugate solutions,  $r = \alpha \pm i\beta$ . Then the solution is:

$$y_h(t) = e^{\alpha t} (C_1 \cos(\beta t) + C_2 \sin(\beta t))$$

If  $a, b, c > 0$ , then  $\alpha = -(b/2a) < 0$ . In the case of complex roots, the system is said to the UNDERDAMPED. If  $\alpha = 0$  (this occurs when there is no damping), we get pure periodic motion, with period  $2\pi/\beta$  or circular frequency  $\beta$ .

## Solving the more general case

We had two methods for solving the more general equation:

$$y'' + p(t)y' + q(t)y = 0$$

but each method relied on already having one solution,  $y_1(t)$ . Given that situation, we can solve for  $y_2$  (so that  $y_1, y_2$  form a fundamental set), by one of two methods:

- By use of the Wronskian: There are two ways to compute this,

$$- W(y_1, y_2) = Ce^{-\int p(t) dt} \text{ (This is from Abel's Theorem)}$$

$$- W(y_1, y_2) = y_1 y_2' - y_2 y_1'$$

Therefore, these are equal, and  $y_2$  is the unknown:  $y_1 y_2' - y_2 y_1' = Ce^{-\int p(t) dt}$

- Reduction of order, where  $y_2 = v(t)y_1(t)$ . Now substitute  $y_2$  into the DE, and use the fact that  $y_1$  solves the homogeneous equation, and the DE reduces to:

$$y_1 v'' + (2y_1' + p y_1)v' = 0$$

## Finding the particular solution.

Our two methods were: Method of Undetermined Coefficients and Variation of Parameters.

### Method of Undetermined Coefficients

This method is motivated by the observation that, a linear operator of the form  $L(y) = ay'' + by' + cy$ , acting on certain classes of functions, returns the same class. In summary, the table from the text:

if $g_i(t)$ is:	The ansatz $y_{p_i}$ is:
$P_n(t)$	$t^s(a_0 + a_1 t + \dots + a_n t^n)$
$P_n(t)e^{\alpha t}$	$t^s e^{\alpha t}(a_0 + a_1 t + \dots + a_n t^n)$
$P_n(t)e^{\alpha t} \sin(\mu t)$ or $\cos(\mu t)$	$t^s e^{\alpha t} ((a_0 + a_1 t + \dots + a_n t^n) \sin(\mu t) + (b_0 + b_1 t + \dots + b_n t^n) \cos(\mu t))$

The  $t^s$  term comes from an analysis of the homogeneous part of the solution. That is, multiply by  $t$  or  $t^2$  so that no term of the ansatz is included as a term of the homogeneous solution.

### Variation of Parameters:

Given  $y'' + p(t)y' + q(t)y = g(t)$ , with  $y_1, y_2$  solutions to the homogeneous equation, we write the ansatz for the particular solution as:  $y_p = u_1 y_1 + u_2 y_2$ . Substitute this in, and from our analysis, we saw that  $u_1, u_2$  were required to solve the following, which can be done using Cramer's Rule:

$$\begin{aligned} u_1' y_1 + u_2' y_2 &= 0 \\ u_1' y_1' + u_2' y_2' &= g(t) \end{aligned} \quad \Rightarrow \quad u_1' = \frac{-y_2 g}{W(y_1, y_2)} \quad u_2' = \frac{y_1 g}{W(y_1, y_2)}$$

## Analysis of the Oscillator Model (3.7-3.8)

Given

$$mu'' + \gamma u' + ku = F(t)$$

we should be able to determine the constants from a given setup for a spring-mass system. Once that's done, be able to analyze the spring-mass system in some particular cases:

1. Unforced (The homogeneous equation,  $F(t) = 0$ )
  - (a) No damping: Natural frequency is  $\sqrt{k/m}$
  - (b) With damping: Underdamped, Critically Damped, Overdamped

### 2. Periodic Forcing<sup>1</sup>

- (a) With no damping: Determine when Beating and Resonance occur.

$$u'' + \omega^2 u = F \cos(\omega_0 t)$$

“Beating” occurs when  $\omega$  is close to  $\omega_0$ .

The circular frequency for one beat is  $|\omega_0 - \omega|$ . The amplitude of one beat:  $2F/(\omega_0^2 - \omega^2)$ .

“Resonance” occurs when  $\omega = \omega_0$ . Resonance forces the solution to become unbounded (can be very bad in the physical world!)

- (b) With damping: We changed the model to make our computations a bit easier:

$$u'' + pu' + qu = \cos(\omega t)$$

Then with  $y_p = Ae^{i\omega t}$ , we found that

$$A = \frac{1}{(q - \omega^2) + ip\omega} = \frac{1}{\alpha + \beta i}$$

Given this, we found that the amplitude  $R$  and phase angle  $\delta$  of the forced response (also known as the particular part of the solution) is given by:

$$R = \frac{1}{|\alpha + \beta i|} \quad \delta = \tan^{-1}(\beta/\alpha)$$

Given this, be able to determine  $q$  or  $\omega$  that will maximize the amplitude  $R$  (by differentiating, setting to zero.) Using that value of  $\omega$  in the physical world can result in Resonance (blowing up the wine glass or a bridge!)

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<sup>1</sup>The more general case of forcing we would use the Method of Undetermined Coefficients to solve.