Outline- Section 3.8

- Full model: $mu'' + \gamma u' + ku = F(t)$.
- Last time: Generic F(t) (Method of Undet Coeffs or Var of Params)
- This time: $F(t) = \cos(\omega t)$ or $\sin(\omega t)$ (Periodic forcing).

Side remark on Vocab:

- 1. Transient part of the solution part of soln \rightarrow zero as $t \rightarrow \infty$.
 - $m, \gamma, k > 0$ then $y_h(t) \rightarrow 0$.
- 2. Forced Response part of soln remaining. Also called the steady state solution.

Today we look at the forced response...

But what's new

Analyze in the case of our model:

- ► No Damping, Periodic Forcing.
- Damping and Periodic Forcing.

No Damping, Periodic Forcing

Model

$$mu'' + ku = A\cos(\omega t) \quad \Rightarrow \quad u'' + \omega_0^2 u = F_0\cos(\omega t)$$

The homogeneous part of the solution can be expressed as:

$$u_h(t) = C_1 \cos(\omega_0 t) + C_2 \sin(\omega_0 t)$$

For the particular solution, two subcases: $\omega \neq \omega_0$ and $\omega = \omega_0$

Forcing and Natural Frequencies are Distinct

$$u'' + \omega_0^2 u = F_0 \cos(\omega t)$$

If $\omega_0 \neq \omega$, then ansatz: $y_p = Ae^{i\omega t}$ and $y'_p = i\omega Ae^{i\omega t}$ and $y''_p = -\omega^2 Ae^{i\omega t}$. $Ae^{i\omega t}(-\omega^2 + \omega_0^2) = F_0 e^{i\omega t} \Rightarrow A = \frac{F_0}{\omega_0^2 - \omega^2}$

so the overall solution is:

$$y(t) = C_1 \cos(\omega_0 t) + C_2 \sin(\omega_0 t) + \frac{F_0}{\omega_0^2 - \omega^2} \cos(\omega t)$$

To simplify the analysis, we'll also assume y(0) = y'(0) = 0(Exercise 1)

$$C_1 = -\frac{F_0}{\omega_0^2 - \omega^2} \qquad C_2 = 0$$

In this case,

$$y(t) = \frac{F_0}{\omega_0^2 - \omega^2} \left(\cos(\omega t) - \cos(\omega_0 t) \right)$$

To "simplify" our analysis, we rewrite this difference as a product.

Write solution as a product

Using some trig (See pg 213):

$$\frac{2F_0}{\omega_0^2 - \omega^2} \sin\left(\frac{(\omega_0 - \omega)t}{2}\right) \sin\left(\frac{(\omega_0 + \omega)t}{2}\right)$$

If $\omega\approx\omega_{\rm 0},$ the first term is a larger and longer wave.

$$\pm \frac{2F_0}{\omega_0^2 - \omega^2} \sin\left(\frac{(\omega_0 - \omega)t}{2}\right)$$

Let's see a graph...

Graph of the solution, $\omega = 0.8, \omega_0 = 1$



This is Beating...

Circular frequency of a single beat (half the frequency of the sine): $|\omega_0 - \omega|$.

$$\frac{2F_0}{\omega_0^2 - \omega^2} \sin\left(\frac{(\omega_0 - \omega)t}{2}\right) \sin\left(\frac{(\omega_0 + \omega)t}{2}\right)$$

If $\omega \approx \omega_0$, the first term is a larger and longer wave.

$$\pm \frac{2F_0}{\omega_0^2 - \omega^2} \sin\left(\frac{(\omega_0 - \omega)t}{2}\right)$$

The "envelope" gets longer and longer with larger and larger amplitude....

What happens at $\omega = \omega_0$? Something known as **resonance**.

Several ways of getting the particular solution:

- 1. Start Method of Undet Coeffs over again. Ansatz: $u_p = Ate^{i\omega_0 t}$.
- 2. Use the previous solution, and take the limit as $\omega \rightarrow \omega_0$.

Take limit (l'Hospital's Rule)

$$\lim_{\omega \to \omega_0} \frac{F_0 \left(\cos(\omega t) - \cos(\omega_0 t) \right)}{\omega_0^2 - \omega^2} =$$
$$\lim_{\omega \to \omega_0} \frac{-F_0 t \sin(\omega t)}{-2\omega} = \frac{F_0}{2\omega_0} t \sin(\omega_0 t)$$

Now "blows up", or becomes unbounded... This is resonance.

Resonance occurs when the forcing freq matches the natural freq.

Summary so far...

Given $u'' + \omega_0^2 u = F_0 \cos(\omega t)$

1. If $\omega \approx \omega_0$, we get **Beating**. The (circ) freq of one beat is $|\omega_0 - \omega|$.

2. If $\omega = \omega_0$, then **Resonance**

(Pause for the video)

Full model, Periodic Forcing

$$mu'' + \gamma u' + ku = F_0 \cos(\omega t)$$

Slightly altered:

$$u'' + pu' + qu = \cos(\omega t)$$

The characteristic equation has roots:

$$r=\frac{-p\pm\sqrt{p^2-4q}}{2}, \quad p>0$$

Implies that, always, we have

$$u_p(t) = A\cos(\omega t) + B\sin(\omega t)$$

No solution is completely unbounded...

Does a form of resonance persist?

Given a fixed p, q in the model, is it possible to **tune** the forcing frequency to **maximize** the amplitude of the forced response?

Answer: It is! (Break a wine glass with your voice)

Full model, Periodic Forcing

$$u'' + pu' + qu = \cos(\omega t)$$

Ansatz $y_p = Ae^{i\omega t}$ and $y'_p = Ai\omega e^{i\omega t}$ and $y''_p = -A\omega^2 e^{i\omega t}$.
Then

$$Ae^{i\omega t}(-\omega^2+i\omega p+q)=e^{i\omega t}$$

so that

$$A = \frac{1}{(q - \omega^2) + i\omega p}$$

and the particular solution is

$$u_p = \operatorname{Re}\left(A \mathrm{e}^{i\omega t}\right)$$

Given

$$u_p(t) = \operatorname{Real}\left(\frac{1}{(q-\omega^2)+i\omega p}e^{i\omega t}\right) = R\cos(\omega t - \delta)$$

(See Complexification Handout) Amplitude R and phase angle δ for u_p are given by:

$$R(\omega) = rac{1}{|(q-\omega^2)+i\omega p|} \qquad \delta = an^{-1}\left(rac{\omega p}{q-\omega^2}
ight)$$

We observe that R (the amplitude of u_p) is a function of ω .

Can we maximize R?

Set the derivative to zero and solve for $\omega ...$ NOTE: If

$$R = rac{1}{\sqrt{f(\omega)}} \quad \Rightarrow \quad R' = rac{1}{2}(f(\omega))^{-1/2}f'(\omega)$$

Therefore, if we solve for R'=0, we only need $f'(\omega)=0$

Continuing - Where is *R* at its maximum?

$$R = \frac{1}{|(q - \omega^2) + i\omega p|} = \frac{1}{\sqrt{(q - \omega^2)^2 + \omega^2 p^2}}$$
$$f(\omega) = (q - \omega^2)^2 + p^2 \omega^2 \quad \Rightarrow \quad \frac{df}{d\omega} = 2(q - \omega^2)(-2\omega) + p^2 \cdot 2\omega = 0$$
Solving for only the positive ω , we get $\omega = \sqrt{\frac{2q - p^2}{2}}$.

Numerical Example

Find the forced response to

$$u''+u'+2u=\cos(2t)$$

SOLUTION: Ansatz is Ae^{2it}.Substitute and factor:

$$Ae^{2it}(-4+2i+2) = e^{2it} \Rightarrow A = \frac{1}{-2+2i}$$

Therefore, the amplitude and phase will be

$$R = \frac{1}{|-2+2i|} = \frac{1}{\sqrt{4+4}} = \frac{1}{\sqrt{8}} \qquad \delta = \arg(-2+2i) = \frac{3\pi}{4}$$

Therefore,

$$u_{p} = \frac{1}{2\sqrt{2}} \cos\left(2t - \frac{3\pi}{4}\right)$$

Numerical Example 2

Find ω that maximizes the amplitude of the forced response to:

$$u'' + u' + 2u = \cos(\omega t)$$

SOLUTION: Ansatz is $Ae^{i\omega t}$. Substitute and factor:

$$Ae^{i\omega t}(-\omega^2 + i\omega + 2) = e^{i\omega t} \quad \Rightarrow \quad A = \frac{1}{(2-\omega^2) + i\omega}$$

Therefore, the amplitude is

$$R = \frac{1}{|(2 - \omega^2) + i\omega|} = \frac{1}{\sqrt{(2 - \omega^2)^2 + \omega^2}} = \frac{1}{\sqrt{f(\omega)}}$$

Therefore, $R'(\omega) = 0$ when $f'(\omega) = 0$, which is computed:

$$f(\omega) = (2 - \omega^2)^2 - \omega^2 \quad \Rightarrow \quad f'(\omega) = 2(2 - \omega^2)(-2\omega) + 2\omega = 0$$

Solving for ω , we get $\omega = \sqrt{\frac{3}{2}}$



 $0, \frac{1}{2}\sqrt{6}, -\frac{1}{2}\sqrt{6}$

4 5

Numerical Example 3

Find ω that maximizes the amplitude of the forced response to:

$$u''+\frac{1}{10}u'+2u=\cos(\omega t)$$

We'll just compare the maximum amplitude:

$$R = \frac{1}{|(2 - \omega^2) + i\omega/10|} = \frac{1}{\sqrt{(2 - \omega^2)^2 + \omega^2/100}}$$



What have we shown?

- With damping and periodic forcing, no unbdd solns.
- However, are able to "tune" the freq of the forcing function to maximize the response.
- The smaller the relative size of the damping, the larger the maximum amplitude.
- This has very important engineering implications (Go to video!)